Treatment of Discontinuities

- Today, we shall look at the problem of dealing with discontinuities in models.
- Models from engineering often exhibit discontinuities that describe situations such as switching, limiters, dry friction, impulses, or similar phenomena.
- The modeling environment must deal with these problems in special ways, since they influence strongly the numerical behavior of the underlying differential equation solver.



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Numerical Differential Equation Solvers

- Most of the *differential equation solvers* that are currently on the market operate on *polynomial extrapolation*.
- The value of a state variable *x* at time *t*+*h*, where *h* is the current *integration step size*, is approximated by fitting a *polynomial of nth order* through known supporting values of *x* and *dx/dt* at the current time *t* as well as at past instances of time.
- The value of the extrapolation polynomial at time *t*+*h* represents the approximated solution of the differential equation.
- In the case of *implicit integration algorithms*, the state derivative at time *t*+*h* is also used as a supporting value.



Examples

Explicit Euler Integration Algorithm of 1st Order:

 $x(t+h) \approx x(t) + h \cdot \dot{x}(t)$

Implicit Euler Integration Algorithm of 1st Order:

 $x(t+h) \approx x(t) + h \cdot \dot{x}(t+h)$

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Discontinuities in State Equations

- *Polynomials* are always *continuous and continuously differentiable functions*.
- Therefore, when the *state equations* of the system:

 $\dot{x}(t) = f(x(t),t)$

exhibit a discontinuity, the polynomial extrapolation is a very poor approximation of reality.

• Consequently, *integration algorithms* with a fixed step size exhibit a large *integration error*, whereas integration algorithms with a variable step size reduce the step size dramatically in the vicinity of a discontinuity.



Integration Across Discontinuities

- An integration algorithm of variable step size reduces the step size at every discontinuity.
- After passing the discontinuity, the step size is only slowly enlarged again, as the integration algorithm cannot distinguish between a *discontinuity* on one hand and a *point of large local stiffness* (with a large absolute value of the derivative) on the other.



The State Event

• These problems can be avoided by telling the integration algorithm explicitly, when and where discontinuities are contained in the model description.

Example: Limiter Function



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Event Handling I



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Event Handling II



Step size as function of time without event handling

Step size as function of time with event handling

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Representation of Discontinuities

 $f = \text{ if } x < xm \text{ then } fm \text{ else if } x < xp \text{ then } m^*x \text{ else } fp ;$

- In *Modelica*, discontinuities are represented as *if-statements*.
- In the process of translation, these statements are transformed into correct *event descriptions* (sets of *models with switching conditions*).
- The modeler does not need to concern him- or herself with the mechanisms of event descriptions. These are hidden behind the *if-statements*.



Problems

• The modeler needs to take into account that the discontinuous solution is temporarily left during iteration.

$$q = \sqrt{|\Delta p|}$$

$$\Delta p = p_1 - p_2; abs \Delta p = if \Delta p > 0 \text{ then } \Delta p \text{ else } -\Delta p; q = sqrt(abs \Delta p);$$

may be dangerous, since $abs \Delta p$ can become temporarily negative.

$$\Rightarrow \Delta p = p_1 - p_2;$$

$$abs \Delta p = \text{noEvent}(\text{ if } \Delta p > 0 \text{ then } \Delta p \text{ else } -\Delta p);$$

$$q = \operatorname{sqrt}(abs \Delta p);$$

solves this problem.

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The "noEvent" Construct

 $\Delta p = p_1 - p_2;$ $abs \Delta p = noEvent(if \Delta p > 0 then \Delta p else - \Delta p);$ $q = sqrt(abs \Delta p);$

- The *noEvent construct* has the effect that *if-statements* or *Boolean expressions*, which normally would be translated into simulation code containing correct event handling instructions, are handed over to the integration algorithm untouched.
- Thereby, management of the simulation across these discontinuities is left to the step size control of the numerical Integration algorithm.



Multi-valued Functions I

• The language constructs that have been introduced so far don't suffice to describe *multi-valued functions*, such as the *dry hysteresis function* shown below.



- When *x* becomes greater than x_p , *f* must be switched from f_m to f_p .
- When x *becomes smaller* than x_m , f must be switched from f_p to f_m .

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Multi-valued Functions III





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The Electrical Switch I



When the switch is *open*, the current is i=0. When the switch is *closed*, the voltage is u=0.

0 =**if** *open* **then** *i* **else** *u* ;

The *if-statement* in *Modelica* is *a-causal*. It is being sorted together with all other statements.

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The Electrical Switch II

Possible Implementation:

Switch open: s = 1Switch closed: s = 0

$$\implies 0 = s \cdot i + (1 - s) \cdot u$$

Switch open:

Sf
$$| f = 0$$

Switch closed:

Se
$$e = 0$$

$$\Rightarrow \qquad s \longrightarrow Sw \frac{e}{f}$$

The causality of the switch element is a function of the value of the control signal s.

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The Ideal Diode I



When u < 0, the switch is open. No current flows through.

When u > 0, the switch is closed. Current may flow. The ideal diode behaves like a short circuit.

open = u < 0; 0 = if open then i else u;





The Ideal Diode II

• Since current flowing through a diode cannot simply be interrupted, it is necessary to slightly modify the diode model.

open = $u \le 0$ and not i > 0; θ = if open then i else u;

• The variable *open* must be declared as *Boolean*. The value to the right of the Boolean expression is assigned to it.



The Friction Characteristic I

- More complex phenomena, such as friction characteristics, must be carefully analyzed case by case.
- The approach is discussed here by means of the friction example.



When $v \neq 0$, the friction force is a function of the velocity.

When v = 0, the friction force is computed such that the velocity remains 0.



The Friction Characteristic II

• We distinguish between five situations:

v = 0 a = 0	<u>Sticking</u> : The friction force compensates the sum of all forces attached, except if $ \Sigma f > R_0$.
v > 0	<u>Moving forward</u> : The friction force is computed as: $f_B = R_v \cdot v + R_m.$
v < 0	<u>Moving backward</u> : The friction force is computed as: $f_B = R_v \cdot v - R_m$.
v = 0 a > 0	Beginning of forward motion:The friction force is computed as: $f_B = R_m$
v = 0 a < 0	Beginning of backward motion:The friction force is computed as: $f_B = -R_m$



The State Transition Diagram

• The set of events can be described by a *state transition diagram*.



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The Friction Model I

model Friction;
parameter Real R0, Rm, Rv;
parameter Boolean ic=false;
Real fB, fc;
Boolean Sticking (final start = ic);
Boolean Forward (final start = ic), Backward (final start = ic);
Boolean StartFor (final start = ic), StartBack (final start = ic);
fB = if Forward then Rv*v + Rm else
 if Backward then Rv*v - Rm else
 if StartFor then Rm else

if *StartBack* **then** *-Rm* **else** *fc*;

0 = **if** *Sticking* **or initial() then** *a* **else** *fc*;

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The Friction Model II

```
when Sticking and not initial() then
    reinit(v,0);
end when;
```

<i>Forward</i> =	initial()	and $v > 0$ or
	<pre>pre(StartFor)</pre>	and $v > 0$ or
	pre (Forward)	and not <i>v</i> <= <i>0</i> ;
Backward =	initial()	and $v < 0$ or
	<pre>pre(StartBack)</pre>	and $v < 0$ or
	<pre>pre(Backward)</pre>	and not $v \ge 0$;

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The Friction Model III

 $\begin{array}{ll} StartFor &= \mathbf{pre}(Sticking) & \text{and } fc > R0 \text{ or} \\ & \mathbf{pre}(StartFor) & \text{and not} \ (v > 0 \text{ or } a <= 0 \text{ and not} \ v > 0); \\ StartBack &= \mathbf{pre}(Sticking) & \text{and } fc < -R0 \text{ or} \\ & \mathbf{pre}(StartBack) \text{ and not} \ (v < 0 \text{ or } a >= 0 \text{ and not} \ v < 0); \\ Sticking &= \mathbf{not} \ (Forward \text{ or } Backward \text{ or } StartFor \text{ or } StartBack); \end{array}$

end Friction;

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